Sheet material feeding apparatus

A sheet material feeding apparatus includes a feed-out device (K1, K2) that brings the stacked sheet materials (1) conveyed on a floor belt (11) into contact with a pickup roller (2a, 2b) and feeds the sheet materials out in a direction crossing the conveying direction of the stacked sheet materials (1) by rotation of the pickup roller (2a, 2b), and an inclination correcting device (6) provided in an upstream of the feed-out device (K1, K2) in its sheet material feeding direction and including an upper roller (6a) located at a position higher than that of the pickup roller (2a, 2b), that brings an upper side of the stacked sheet materials (1) by rotation of the pickup roller (2a, 2b), and an inclination correcting device (6) provided in an upstream of the feed-out device (K1, K2) in its sheet material feeding direction and including an upper roller (6a) located at a position higher than that of the pickup roller (2a, 2b), that brings an upper side of the stacked sheet materials (1) by rotation of the pickup roller (2a, 2b), and an inclination correcting device (6) provided.
Description

[0001] The present invention relates to a sheet material feeding apparatus that feeds a sheet material such as a postal matter.

[0002] As a sheet material feeding apparatus of this kind, there is known a type in which sheet materials are placed in a standing position on a floor belt and they are conveyed to a feed-out unit as the floor belt runs. At the feed-out unit, sheet materials are fed out one by one as feed-out roller or feed-out belt rotates.

[0003] However, in some cases, stacked sheet materials are conveyed in an inclined state as relative to the feed-out unit as being affected so for one reason or another or the stacked sheet materials are already set in an inclined state on the floor belt before they are conveyed. In such cases, the sheet materials are not evenly brought into contact with the feed-out roller or feed-out belt, and thus an uneven contacting state is created. When the sheet materials are fed out in this state, such problems occur that sheets are skewed and continuous feeding out of sheets is interrupted.

[0004] As a solution to this, a conventional technique was developed as disclosed in, for example, Jpn. Pat. Appln. KOKAI Publication No. 2002-68490, in which the floor belt is divided and a lever is provided to convey sheet materials while detecting the clearance between sheet materials, which is roughly equivalent to the inclining state, to realize stable feeding out of the materials.

[0005] However, with the above-described method, a mechanism for operating the lever is necessary, which makes the structure of the device complicated, and there is such a drawback that sheet material may be damaged when the lever fails to function properly.

[0006] The present invention has been proposed in consideration of the above-described circumstances of the conventional technique and its object is to provide a sheet material feeding apparatus with a simple structure, which can correct the inclination of stacked sheet materials without the possibility of damaging any of the materials.

[0007] According to an aspect of the present invention, there is provided a sheet material feeding apparatus comprising: a conveying device that conveys stacked sheet materials in a stacking direction in a standing state; a feed-out device that brings the stacked sheet materials conveyed by the conveying device into contact with a feed-out roller and feeds the sheet materials out in a direction crossing the conveying direction of the stacked sheet materials by rotation of the feed-out roller; a guide unit that guides the sheet materials fed out by the feed-out device along a guide surface such as to project a part of the feed-out roller from the guide surface; and an inclination correcting device provided in an upstream of the feed-out device in its sheet material feeding direction and including a roller member located at a position higher than that of the feed-out roller, that brings an upper side of the stacked sheet materials conveyed towards the feed-out device into contact with the roller member to correct the inclination of the stacked sheet materials.

[0008] According to the above-described embodiment of the present invention, the inclination of stacked sheet materials can be corrected merely by bringing the sheet materials into contact with the roller member of the inclination correcting device. Thus, the invention has a simple structure and does not damage sheet materials. Further, the sheet materials are handled to be displaced from each other to be fed out, and thus they can be separated surely one by one.

[0009] This summary of the invention does not necessarily describe all necessary features so that the invention may also be a sub-combination of these described features.

[0010] The invention can be more fully understood from the following detailed description when taken in conjunction with the accompanying drawings, in which:

FIG. 1 is a plan view showing a sheet material feeding apparatus according to the first embodiment of the present invention;
FIG. 2 is a front view showing a mechanism of correcting inclination of stacked sheet materials in the sheet material feeding apparatus;
FIG. 3 is a diagram showing a force applied to stacked sheet materials by the inclination correcting mechanism;
FIG. 4 is a front view showing an inclination correcting mechanism according to the second embodiment of the present invention;
FIG. 5 is a plan view showing a set of inclination correcting mechanisms according to the third embodiment of the present invention;
FIG. 6 is a plan view showing an inclination correcting mechanism according to the fourth embodiment of the present invention;
FIG. 7 is a front view of the inclination correcting mechanism shown in FIG. 6;
FIG. 8 is a front view showing a moving mechanism of the inclination correcting mechanism shown in FIG. 6;
FIG. 9 is a plan view of moving mechanism shown in FIG. 8;
FIG. 10 is a plan view showing an inclination correcting mechanism according to the fifth embodiment of the present invention; and
FIG. 11 is a front view of the inclination correcting mechanism shown in FIG. 10.

[0011] Embodiments of the present invention will now be described in detail with reference to accompanying drawings.

[0012] FIG. 1 is a plan view schematically showing the structure of a sheet material feeding apparatus according to the first embodiment of the present invention.

[0013] This sheet material feeding apparatus includes
In the feeding direction of sheet materials, a floor belt 11 serving as a conveying device that conveys stacked sheet materials 1 placed thereon in a standing position. A rear end side of the stacked sheet materials 1 with respect to the direction of the materials being conveyed by the floor belt 11 is held with a pressure by a backup plate 12. The backup plate 12 is moved along a backup plate shaft 13.

0014 In the downstream of the conveying direction of the stacked sheet materials 1, the first and second feeding mechanisms K1 and K2 are provided as a feed-out device. Sheet materials are fed by the first and second feeding mechanisms K1 and K2 in a direction that crosses the direction of conveying the stack sheet materials 1. Further, a guide plate 15 serving as a guide member is provided along the direction of feeding sheet materials, and while conveying the sheet materials, they are guided along a guide surface 15a of the guide plate 15.

0015 The first feeding mechanism K1 includes upper and lower pick-up rollers 2a and 2b and these pick-up rollers 2a and 2b are each supported by a pick-up arm 3. A drive motor 4 is connected to the pick-up roller 2 via a timing belt 2. The second feeding mechanism K2 has a similar structure to that of the first feeding mechanism 1, and therefore similar parts are designated by the same reference symbols and the explanations therefor will be omitted.

0016 In the upstream of the sheet material feeding direction of the second feeding mechanism K2, an inclination correcting mechanism 6 is provided as an inclination correcting device to correct the inclination of the stacked sheet materials 1, which will be described in detail later.

0017 The inclination correcting mechanism 6 is rotatably supported by one end of a support arm 7 serving as a support device. The support arm 7 is pivotally supported on a pivot 7a by its middle portion and the other end thereof is urged by a spring member 8. With the urging force of the spring member 8, the inclination correcting mechanism 6 is brought into contact with a pressure on the upper side of the rear end side of the stacked sheet materials in the feeding direction.

0018 In the feeding direction of sheet materials, a feed roller 9 is provided to feed sheet materials to the downstream, and a reverse roller 10 is provided for the feed roller 9 in order to separate materials from the next sheet on by applying a counter-rotating torque thereto.

0019 FIG. 2 is a front view showing the inclination correcting mechanism 6 described above.

0020 The inclination correcting mechanism 6 includes a support shaft 14. An upper roller 6a serving as an upper end roller member is provided on an upper end portion of the support shaft 14, a middle roller 6b is mounted in a middle portion thereof, and a lower roller 6c is mounted in a lower portion thereof. The support shaft 14 is inclined such that the upper end side thereof is set close to the stacked sheet materials 1, whereas the lower end side is distant from the stacked sheet materials 1. The rollers 6a to 6c are arranged to be in parallel with each other, and all have the same diameter.

0021 The inclining angle of the support shaft 14 is set to 4°, and the heights of the rollers 6a to 6c, that is, the height to a corner portion that is brought into contact with sheet materials in each, are 140 mm for the upper roller 9a, 90 mm for the middle roller 6b and 20 mm for the lower roller 6c.

0022 The amount of projection of the upper roller 6a and middle roller 6b from the guide plate 15 is set a predetermined amount larger than the projecting amount of the pickup roller 2 from the guide plate 15. With this arrangement, as stacked sheet materials becomes higher and more easily inclined, the inclination of the stack is more corrected as the stack abuts against the rollers of the upper side.

0023 It should be noted here that if the lower roller 6c is set to project in a similar manner to those of the rollers 6a and 6b on the upper side, such an arrangement in some cases prevent stacked sheet materials from being brought into contact with the lower pickup roller 2b. In order to avoid this, the projecting amount of the lower roller 6c is set smaller than the projecting amount of the lower pickup roller 2b.

0024 Next, the feeding operation of the sheet material feeding apparatus will now be described.

0025 First, the stacked sheet materials 1 are set in a standing position on the floor belt 11, and then the floor belt 11 and backup plate 12 are moved in the direction indicated in FIG. 1 to convey the stacked sheet materials 1. While this operation, if the stacked sheet materials 1 are conveyed in an inclined state, for example, while being inclined such that the upper side of the stack is close to the pickup roller 2b, whereas the lower side thereof is distant from the pickup roller 2b, as shown in FIG. 2, the upper side of the stack of the sheet materials 1 is brought into contact with a pressure with the upper roller 6a so that the upper side is pushed back. In this manner, the inclination of the stacked sheet materials 1 is corrected so that they stand straight up. After the inclination is corrected as described, the stacked sheet materials 1 are set to abut against the pickup rollers 2a and 2b. Sheet materials abutting against the pickup rollers 2a and 2b are fed to the feed roller 9 as the pickup rollers 2a and 2b rotate. While this operation, the rollers 6a and 6b are rotated to follow the rotation of the pickup rollers 2a and 2b via a respective sheet material, thereby making it possible to reduce the load during the feeding. The fed sheet materials are separated one by one from each other by the rotations of the feed roller 9 and reverse roller 10, and they are conveyed to the downstream as indicated by a white arrow in FIG. 1.

0026 FIG. 3 illustrates the details of the operation of the upper roller 6a of the inclination correcting mechanism 6.

0027 Due to the urging force of the spring member
8, there is created a force applied to the upper roller 6a to move in the directed indicated by arrow F. The upper side of the rear end of the stacked sheet materials 1 is pushed against the upper roller 6a. Here, the direction of the force propagated to the stacked sheet materials 1 changes depending upon with which part of the upper roller 6a the stacked sheet materials 1 are brought into contact. More specifically, at a position fa which corresponds to a relatively beginning stage where the stack of sheet materials 1 are brought into contact with the upper roller 6a, the component of the force that acts in the stacking direction of the sheet materials is larger than the component of the force that acts in the planer direction of the sheet materials. On the other hand, at a position fb which corresponds to a relatively ending stage of the contact between the stack of sheet materials 1 and the upper roller 6a, the component of the force that acts in the planer direction of the sheet materials is larger than the component of the force that acts in the stacking direction of the sheet materials.

[0028] It should be noted here that the component of the force that acts in the stacking direction of the sheet materials serves as a force to support the sheet materials being conveyed in the inclined state, whereas the component of the force that acts in the planer direction of the sheet materials serves as a force to feed the sheet materials.

[0029] Therefore, the stacked sheet materials 1 are shifted from the beginning state where they are pushed in the direction of correcting its inclination to the state where they are fed in the feeding direction. This transition of the state is very convenient to maintain the continuous sheet material feeding operation.

[0030] Further, as the component of the force applied on the stacked sheet materials 1 shifts its direction, such an effect of separating the stacked sheet materials from each other, which is so called "handling effect", is obtained. Thus, the load resistance due to the tightly stacking of the sheet materials, can be reduced.

[0031] As described, due to the effect obtained by the arrangement of the inclination correcting mechanism 6, a stable feeding out operation can be carried out even if stacked sheet materials, for example, fall.

[0032] In this embodiment, the outer diameter of the upper roller 6a is set to φ 40 mm, the radius of pivoting is set to 40 mm, and the projecting amount from the guiding plate 15 is set to 20 mm ± 2 mm.

[0033] Further, the pivotable range θ of the upper roller 6a is set to 35° to 65°, and the value of F is set to 7.2N (0.8 kgf) to 14.7N (1.5 kgf), which is set by arranging the spring member 8 appropriately and selecting an appropriate spring constant.

[0034] Under the above-described conditions, sample envelopes were prepared and a continuous feeding test was carried out.

[0035] The sample envelopes were those of a size C5 (229 mm × 162 mm), and different contents were enclosed in the envelopes, with such an adjustment of the ratio in the number of envelopes so that the weights of the envelope vary from 10g to 50g, at an average of 25g.

[0036] When the apparatus was operated with a control order of feeding the sheet materials at a conveying speed of 3.6 m/s and a constant gap of 100 mm, which is equivalent to 38.2k envelopes/h), it was possible to stably and continuously carry out the feeding out operation at an average gap of 109.4 mm and an accuracy of a standard deviation of 10 mm.

[0037] Further, since the upper roller 6a was set swingable, and therefore a similar effect to the above can be obtained regardless of the length of the sheet materials as long as it is within the swinging range.

[0038] Actually, the length of the sheet materials was varied from 220 mm to 250 mm and the test was carried out. In each case, it was possible to stably and continuously carry out the feeding out operations at an average gap of 108 to 112 mm and an accuracy of a standard deviation of 6 to 10 mm.

[0039] This embodiment is appropriate for such usage that sheet materials of a length from 220 mm to 250 mm are handled, and it is a feeding out apparatus with an excellent performance in handling, mainly, envelopes of the size C5.

[0040] Here, for sheet materials of a length of less than 220 mm, it is possible to feed out them stably by means of the feeding out force of the pickup roller 2 itself even without the upper roller 6a. Therefore, it is sufficiently possible to handle sheet materials of shorter lengths.

[0041] Further, in this embodiment, the upper, middle and lower rollers 6a to 6c are fixed onto the common arm 7 by the shaft 14, but it is alternatively and perfectly possible to prepare separate arms for the respective rollers in an independent suspension structure.

[0042] FIG. 4 is a front view showing an inclination correcting mechanism 21 according to the second embodiment of the present invention.

[0043] In the second embodiment, the inclination correcting mechanism 21 is provided vertically without being inclined. That is, the support shaft 14 is provided vertically, and an upper roller having a large diameter, a middle roller 17 having a middle diameter and a lower roller 18 having a small diameter are mounted on the support shaft 14 horizontally at predetermined intervals.

[0044] With this structure, as the stacked sheet materials become higher and more easily inclined, they abut more against the upper roller for a better support.

[0045] Similar tests to those of the first embodiment were conducted under such conditions that the diameters of the upper roller 16, middle roller 17 and lower roller 18 were set to φ56 mm, φ50 mm, and φ40 mm, it was possible to stably and continuously carry out the feeding out operations on sheet materials of a length of 220 mm to 250 mm at an average gap of 109 mm to 112 mm and an accuracy of a standard deviation of 7 to 11.

[0046] As described above, it is not necessary to incline the whole inclination correcting mechanism, but a similar effect to that of the first embodiment can be ob-
tained by setting the different diameters for the rollers 16 to 18.

It should be pointed out that as described in connection with the first and second embodiments, the effect of the present invention can be best obtained by arranging the inclination correcting mechanisms 6 and 21 to match the length of sheet materials that are handled mostly. When the length of sheet materials to be handled varies, it is natural to change the arrangement to obtain a similar effect to that.

FIG. 5 is a plan view showing first and second inclination correcting mechanisms 19 and 20 according to the third embodiment of the present invention.

In this embodiment, the first and second inclination correcting mechanisms 19 and 20 are provided in a multiple arrangement along the feeding out direction of sheet materials. With this arrangement, the apparatus becomes able to deal with such a situation that sheet materials with different lengths are handled at the same time.

In the case where the first and second inclination correcting mechanisms 19 and 20 are arranged in a multiple manner, it is necessary to clear the condition that the first inclination correcting mechanisms 19, which is located on the upstream side in the sheet material feeding direction, must be placed such that the projecting amount of the first inclination correcting mechanism 19 from the guide plate 15 is larger than that of the second inclination correcting mechanism 20, which is located on the downstream side.

The first inclination correcting mechanisms 19 is designed to support sheet materials with relatively large sizes, whereas the second inclination correcting mechanism 20 is designed to support sheet materials with relatively small sizes.

Each of the first and second inclination correcting mechanisms 19 and 20 is set at an inclining angle of $4^\circ$ as in the case of the first embodiment. For the first inclination correcting mechanisms 19, there are three rollers, that is, the upper, middle and lower ones are provided, whereas for the second inclination correcting mechanisms 20, there are two rollers, that is, the middle and lower ones, without the upper one, are provided.

The projecting amount of the first inclination correcting mechanism 19 from the guide plate 15 is set larger by $d_1$ (10 mm) as compared to the projecting amount of the pickup roller 2 from the guide plate 15, and a distance $L_1$ from a central portion of the upper roller 19a to the leading end surface of the sheet materials is 238 mm.

Further, the projecting amount of the second inclination correcting mechanism 20 from the guide plate 15 is set larger by $d_2$ (4 mm) as compared to the projecting amount of the pickup roller 2 from the guide plate 15, and a distance $L_2$ from a central portion of the middle roller 20a to the leading end surface of the sheet materials is 218 mm.

The arrangement of the first inclination correcting mechanism 19 is based on a case were large-sized post cards having a length of about 210 mm (widely popular in, for example, Europe) are handled.

The reason why the projecting amount of the first inclination correcting mechanism 19 is set larger that of the second inclination correcting mechanism 20 is to prevent the second inclination correcting mechanism 20 from being brought into contact with a middle portion of a sheet material, especially when the sheet material is of a larger size, which creates a loading resistance when feeding out sheet materials.

Further, the second inclination correcting mechanism 20 is designed for sheet materials having relatively short lengths. In many cases, sheet materials of short lengths do not have such heights for their lengths, and therefore the inclination correcting amount for a stack of such sheet materials may be less to be sufficient. Therefore, it is not necessary to set a large projecting amount for this mechanism.

With the above-described arrangement, the length of the sheet materials was varied from 160 mm to 250 mm and the test was carried out. In each case, it was possible to stably and continuously carry out the feeding out operations at an average gap of 110 to 120 mm and an accuracy of a standard deviation of 8 to 13 in each case.

FIG. 6 is a plan view showing an inclination correcting mechanism 30 according to the fourth embodiment of the present invention, and FIG. 7 shows its front view.

As in the case of the inclination correcting mechanism 6 shown in FIG. 2, the inclination correcting mechanisms 30 of this embodiment includes a support shaft 14, to which an upper roller 31 serving as the first roller, a middle roller 32 and a lower roller 33 serving as the second roller are mounted at an upper end portion, middle portion and lower portion thereof.

The upper, middle and lower rollers 31 to 33 are arranged to be in parallel with each other, and the sizes of their diameters have the relationship of the upper roller 31 < the middle roller 32 < the lower roller 33. In more specific, the outer diameter of the upper roller 31 is φ34 mm, the outer diameter of the middle roller 32 is φ37 mm and the outer diameter of the lower roller 33 is φ43 mm.

The heights of the upper, middle and lower rollers 31 to 33, that is, the heights taken at corner portions that are brought into contact with the sheet materials, are set to 140 mm, 90 mm and 20 mm, respectively.

The radius of pivoting of the support arm 7 is set to 40 mm, and the projecting amount of each of the upper, middle and lower rollers 31 to 33 from the guide plate 15 is set to 20 mm ± 2 mm.

The distance $L$ from an evening surface 34 of the guide plate 34, which evens the edges of the sheet materials, to the center of the support shaft 14 is set to 241 mm ± 2 mm. The radius of an R section 34b of the guide member 34 serving to guide the leading edges of
the stacked sheet materials 1 between a feed roller 9 and a reverse roller 10 is set to 22 mm. [0065] The radius of the lower roller 33 is set substantially equal to the radius of the R section 34b of the guide plate 34.

[0066] The inclination correcting mechanism 30 having the above-described structure was subjected to feeding tests, in which sheet materials of sample envelopes of a size C5 (229 mm × 162 mm), which is most popularly used outside Japan for the usage of direct mails in particular, are continuously fed.

[0067] That is, if the stacked sheet materials 1 are conveyed in an inclined state as shown in FIG. 7, the upper side of the stack of the sheet materials 1 is brought into contact with a pressure with the upper roller 31 and middle roller 32 so that the upper side is pushed back. In this manner, the inclination of the stacked sheet materials 1 is corrected so that they stand straight up.

[0068] During the correction of the inclination, a component of the force that acts in the stacking direction of the sheet materials, and a component of the force that acts in the planer direction of the sheet materials are created. The component that acts in the stacking direction of the sheet materials serves as a force to support the sheet materials being conveyed in the inclined state, whereas the component that acts in the planer direction serves as a force to feed the sheet materials.

[0069] Then, as the component of the force applied on the stacked sheet materials 1 shifts from the stacking direction to the planer direction, such an effect of separating the stacked sheet materials from each other, which is so called "handling effect", is obtained.

[0070] In this embodiment, the outer diameter of the lower roller 33 is set larger than that of the upper roller 31, and substantially equal to the radius of the R section of the guide plate 34. With this structure, the lower roller 33 can apply the feeding force to the sheet materials, and therefore the sheet materials that have been in tight contact with each other can be separated from each other and such a displaced state can be maintained. Thus, the sheet materials can be allowed to enter between the feed roller 9 and reverse roller 10 while being displaced from each other, thereby making it possible to surely separate the sheet materials from each other.

[0071] It should be noted here that in the case where the outer diameter of the upper roller 31 is made larger to be equal to that of the lower roller 33, the phase between the outer diameter of the upper roller 31 and the R section 34b of the guide plate 34 is shifted. More specifically, the distance between the upper roller 31 and the guide plate 34 becomes smaller than 229 mm, which is a length of a C5 size. With this arrangement, the rear end of a sheet material in its feeding-out direction may be caught in the upper roller 31, thereby failing to set displacement between the sheet materials by the upper roller 31. In this case, the sheet materials enter the pressure contact portion between the feed roller 9 and reverse roller 10 without having a displacement of edges between sheet materials, and therefore the frequency of feeding two or more sheet materials in stack erroneously without being separated from each other is increased.

[0072] FIG. 8 is a front view showing a support mechanism 35 of the inclination correcting mechanism described above, and FIG. 9 is a plan view of the support mechanism.

[0073] The figure shows a substrate 36, a mount surface 36a is formed on its upper surface portion, and FIG. 8 is a front view showing a support mechanism 35 of the inclination correcting mechanism described above, and FIG. 9 is a plan view of the support mechanism.

[0074] A shaft 40 is provided to stand at a central portion of a bottom surface of the support bracket 37, and a middle portion of the pivot arm 7 is pivotably mounted to the upper end portion of the shaft 40. The support shaft 14 is mounted to one end of the pivot arm 7, and upper, middle and lower rollers 31 to 33 are mounted to the support shaft 14. The other end of the pivot arm 7 is connected to the support bracket 37 via a spring member 38.

[0075] The inclination correcting mechanism 30 supported as above is moved to change its position in accordance with the size of sheet materials to be fed.

[0076] For example, in the case where sheet materials of a large size are to be fed, the bolts 38 are loosened and the support bracket 37 is slid to move as indicated by an arrow of a solid line to increase the distance between the rollers 31 to 33 and the guide plate 34. Then, the bolts 38 are tightened to fix the support bracket 37. On the other hand, in the case where sheet materials of a small size are to be fed, the support bracket 37 is slid to move as indicated by an arrow of a broken line to decrease the distance between the rollers 31 to 33 and the guide plate 34.

[0077] FIG. 10 is a plan view showing an inclination correcting mechanism 51 according to the fifth embodiment of the present invention, and FIG. 11 is a front view thereof.

[0078] The inclination correcting mechanism 51 includes an inclined support shaft 14 as in the case of the inclination correcting mechanism 30 described above, and a cylindrical roller member 53 is rotatably mounted to the support shaft 14. The roller member 53 is formed such that the diameter thereof gradually decreases from its lower end portion to the upper end portion. The upper and lower portions of the support shaft 14 are supported by pivot arms 56.

[0079] An upper side 53a of the roller member 53 is located at a level higher than that of a pickup roller 2a, and the radius of a lower side 53b is set substantially equal to the radius of an R section 34b of a guide plate 34.
In the inclination correcting mechanism 51 having the above-described structure, if the stacked sheet materials 1 are conveyed by the conveying device (11) into contact with the roller member (6a, 16, 19a, 20b, 31, 53a) to correct the inclination of the stacked sheet materials (1).

The sheet material feeding apparatus according to claim 1, characterized in that the inclination correcting device (6) includes a support shaft (14) inclined to make a predetermined angle with respect to a normal line, the support shaft supporting the roller member (6a) and further supporting a plurality of roller members (6b, 6c) having a same diameter as that of the roller member (6a), located in a lower side of the roller member (6a) and arranged in parallel with each other at predetermined intervals, and the roller member (6a) on the upper side projects much further from the guide unit (15) than the feed-out roller (2a, 2b).

The sheet material feeding apparatus according to claim 1, characterized in that the inclination correcting device (6) includes a vertical support shaft (14), the support shaft (14) supporting the roller member (16) and further supporting a plurality of roller members (17, 18) having diameters different from each other, located in a lower side of the roller member (16) and arranged in parallel with each other at predetermined intervals, and the roller member (16) on the upper side projects much further from the guide unit (15) than the feed-out roller (2a, 2b).

The sheet material feeding apparatus according to claim 1, characterized in that the inclination correcting device (6) includes a roller member (16) and arranged in parallel with each other at predetermined intervals, and the roller member (16) on the upper side projects much further from the guide unit (15) than the feed-out roller (2a, 2b).

The sheet material feeding apparatus according to claim 1, characterized in that the inclination correcting device (6) includes a support shaft (14) inclined to make a predetermined angle with respect to a normal line, that supports the roller member (31) and a roller member (33) located on a lower side of the roller member (31) and having an outer diameter.

A sheet material feeding apparatus characterized by comprising:

1. a conveying device (11) that conveys stacked sheet materials (1) in a stacking direction in a standing state;
2. a feed-out device (K1, K2) that brings the stacked sheet materials (1) conveyed by the conveying device (11) into contact with a feed-out roller (2a, 2b) and feeds the sheet materials out in a direction crossing the conveying direction of the stacked sheet materials (1) by rotation of the feed-out roller (2a, 2b);
3. a guide unit (15) that guides the sheet materials fed out by the feed-out device (K1, K2) along a guide surface (15a) to project a part of the feed-out roller (2a, 2b) from the guide surface (15a); and
4. an inclination correcting device (6, 19, 20, 21, 30, 51) provided in an upstream of the feed-out device (K1, K2) in its sheet material feeding direction and including a roller member (6a, 16, 19a, 20b, 31, 53a) located at a position higher than that of the feed-out roller (2a, 2b), that brings an upper side of the stacked sheet materi-
larger than that of the roller (31), and
the roller member (31) projects from the guide
unit (15) further than the feed-out roller (2a, 2b).

7. The sheet material feeding apparatus according to
claim 1, characterized by further comprising: a sup-
port device (35) that supports the inclination correct-
ing device (30) to be movable in a direction normal
to the conveying direction of the stacked sheet ma-
terials (1).

8. The sheet material feeding apparatus according to
claim 1, characterized in that the inclination cor-
recting device (51) includes a cylindrical roller mem-
ber (53) inclined to make a predetermined angle with
respect to a normal line, and having such a shape
that a diameter thereof gradually decreases from an
upper side (53a) towards a lower side, and the upper
side (53a) of the roller member (53) projects from
the guide unit (15) further than the feed-out roller
(2a, 2b).